

(6)

Stiffness Matrix $[k]$.

Let Consider:

$\omega_1, \omega_2, \omega_3, \dots, \omega_n \rightarrow$ Nodal displacement parameters
or terms as dof

$w_1, w_2, w_3, \dots, w_n \rightarrow$ Corresponding Nodal loads.
acting at dof.

$$\{w^y\} = \begin{Bmatrix} w_1 \\ w_2 \\ \vdots \\ w_n \end{Bmatrix}, \quad \{\omega^y\} = \begin{Bmatrix} \omega_1 \\ \omega_2 \\ \vdots \\ \omega_n \end{Bmatrix}$$

We know that

$$\{w^y\} = [k] \{\omega^y\} \rightarrow ①$$

$w \Rightarrow$ Nodal loads.

We know that, $[k] \Rightarrow$ Stiffness matrix.

Work done, $P =$ Strain energy $\text{as } \omega^y \rightarrow \text{Dof}$

$$\Rightarrow P = \frac{1}{2} w_1 \omega_1 + \frac{1}{2} w_2 \omega_2 + \frac{1}{2} w_3 \omega_3 + \dots + \frac{1}{2} w_n \omega_n$$

$$P = \frac{1}{2} [w_1, w_2, w_3, \dots, w_n] \begin{Bmatrix} \omega_1 \\ \omega_2 \\ \vdots \\ \omega_n \end{Bmatrix}^T \{w^y\}$$

$$= \frac{1}{2} [\{w\}]^T \{\omega^y\} \rightarrow ②$$

$[\] \rightarrow$ Row man

$\{ \ } \rightarrow$ Column man.

sub in ① we get.

*.

$$P = \frac{1}{2} \{ [k] \{ u^* \} \}^T \cdot \{ u^* \}$$

$$\text{stress work} = \frac{1}{2} [k]^T \{ u^* \}^T \{ u^* \}$$

$$P = \frac{1}{2} \{ u^* \}^T [k] \{ u^* \}$$

$$P = \frac{1}{2} \{ u^* \}^T [k] \{ u^* \} \rightarrow ③$$

$[k]$ is a symmetric matrix. so.
 $[k]^T = [k]$

This is strain energy eqn for a struc.

We need to find the exprn for Strain

mat $[k]$.

Let us consider one dimensional element.

$u_1, u_2, u_3, \dots, u_n$ are the degrees of freedom of that element.

We know that:

$$\text{strain } \{ e \} = [B] \{ u^* \}$$

$$\{ e \}^T = (B)^T \{ u^* \}. \rightarrow ④$$

$$e = \frac{du}{dx}$$

$$e = B \cdot u$$

also we know that

$$\text{stress } \{ \sigma \} = [E] \{ e \}$$

$$\{ \sigma \} = [D] \{ e \} \rightarrow ⑤$$

$$[E] = [D] = \text{Young's modul.}$$

Strain energy expn is given by

$$U = \int_V \frac{1}{2} \{e^T f\} \cdot dV \rightarrow 6$$

Subs. e^T & f in above eqn. 6

$$U = \frac{1}{2} V \cdot \sigma e$$

$\sigma \rightarrow$ Strain
 $e \rightarrow$ Strain
 $V \rightarrow$ Vol.

$$U = \int_V \frac{1}{2} [B]^T \{u^*\}^T \cdot [D] \{e\} \cdot dV$$

$$= \frac{1}{2} \{u^*\}^T \int_V [B]^T \cdot [D] \{e\} \cdot dV$$

$$U = \frac{1}{2} \{u^*\}^T \int_V [B]^T \cdot [D] \cdot [B] \cdot \{u^*\} \cdot dV$$

$$= \frac{1}{2} \{u^*\}^T \left[\int_V [B]^T \cdot [D] \cdot [B] \cdot dV \right] \{u^*\} \rightarrow 7$$

Comparing ③ with ⑦

$$\text{③} \Rightarrow P = \frac{1}{2} \{w^*\}^T [K] \{w^*\}$$

we get:

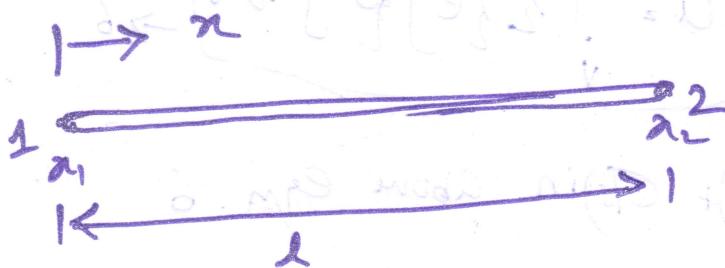
$$[K] = \int_V [B]^T \cdot [D] \cdot [B] \cdot dV$$

~~in~~

~~Strain~~ ~~Strain~~

Derivation of Stiffness matrix for one dimensional Bar elements.

Dimensional Bar elements.



a bar elements with two nodes.

$$\text{Stiffness matrix } [K] = \int [B]^T [D] [B] \cdot dV$$

for 1D elem we know

$$U = N_1 u_1 + N_2 u_2 \quad \text{where } N_1 = \frac{l-x}{l} \quad N_2 = \frac{x}{l}$$

$$[B] = \left[\frac{dN_1}{dx}, \frac{dN_2}{dx} \right]$$

$$[B] = \left[\frac{-1}{l}, \frac{1}{l} \right]$$

$$[B]^T = \left\{ \begin{array}{c} -\frac{1}{l} \\ \frac{1}{l} \end{array} \right\}$$

Sub the value of $[B]^T [B]$ in Stiffness matrix equation

$$[K] = \int_0^l \left\{ \begin{array}{c} -\frac{1}{l} \\ \frac{1}{l} \end{array} \right\} \times E \times \left[\begin{array}{c} -\frac{1}{l} & \frac{1}{l} \end{array} \right] \cdot dV$$

(8)

$$= \int_{x=0}^l \begin{bmatrix} \frac{1}{E^2} & -\frac{1}{E^2} \\ -\frac{1}{E^2} & \frac{1}{E^2} \end{bmatrix} E \cdot dV$$

$$= \int_0^l \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} EA \cdot dx$$

$$= \frac{1}{l^2} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} EA [x]_0^l$$

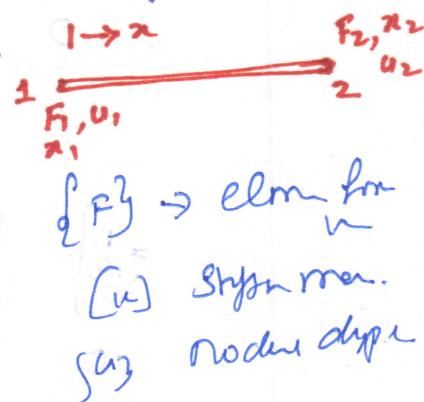
$$= \frac{1}{l^2} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} AE \cancel{x}$$

$$[k] = \frac{AE}{l} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

Properties of a Stiffness matrix.

- ① It is an asymmetric matrix.
- ② The sum of elements in any column is equal to zero.
- ③ It is an unstable element, so the deflection is zero.
- ④ The dimension of the global stiffness matrix $[S]$ is $N \times N$, where N is the no. of nodes.
- ⑤ The diagonal entries are always positive and relatively large when compared to the off-diagonal values in the same row.

Derivation of Finite Element Equations for One dimensional Bar Elements



Derive from Eqn \Rightarrow

$$\{F\} = [K] \{u\}$$

$$[K] = \frac{AE}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$\{F\} = \begin{bmatrix} F_1 \\ F_2 \end{bmatrix}, \{u\} = \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$

$$\begin{bmatrix} F_1 \\ F_2 \end{bmatrix} = \frac{AE}{L} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$

Thus, a FE eqn for 1D. 2 nodal bar elements.